



Damping Properties of Vibrations of Three-Layer Viscoelastic Plate

Safarov Ismail Ibrahimovich¹, Teshayev Muhsin Khudoyberdiyevich²,
Boltayev Zafar Ixtiyorovich², Akhmedov Maqsud Sharipovich²

¹Department of "Mathematics", Tashkent Khimical-Technological Institute, Tashkent, Uzbekistan

²Department of "Mathematics", Bukhara Engineering-Technological Institute, Bukhara, Uzbekistan

Email address:

Muhsin_5@mail.ru (T. M. Khudoyberdiyevich)

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Abstract: The work is devoted to the study of harmonic waves in a hereditarily elastic plate with two viscoelastic coatings, the properties of the material, which are described by the equations of state in integral form. The fractional exponential function of Rabotnov and Koltunov-Rzhanitsyn was chosen as the kernel of the integral operator. Two cases are considered: the case of a stress-strain state symmetric and antisymmetric in the normal coordinate (VAT). In the study of natural oscillations, the properties of those modes that are time-dependent by harmonic law are investigated. For both cases, dispersion equations are derived, which are solved numerically. Asymptotics of the roots of dispersion equations for small and large frequencies are also obtained. The analysis of the obtained solutions made it possible to draw conclusions about the influence of hereditary factors on the behavior of dispersion curves. A comparative analysis of numerical solutions and their asymptotics is carried out.

Keywords: Dispersion Equations, Stress-Strain State, Hereditarily Elastic Layer, Asymptotics

1. Introduction

The increasing need to reduce the vibrations of structural elements caused by loads with a broadband frequency spectrum (for example, aircraft body vibrations) has drawn attention to viscoelastic coatings as a possible solution to this problem. The frequency equation for such systems was obtained from the theory of elasticity, for example [1-3]. In [1] the problem of the propagation of waves of three-layer elastic beams led to a transcendental equation containing hyperbolic functions. The solution of the corresponding transcendental equation for the plates was obtained only for the two lower branches [2, 3]. The calculation of the lower branches was performed using the expansion of transcendental functions into power series, which limits the range of applicability of the results. Another type of solution was obtained in the problem of longitudinal oscillations of a cylindrical rod with a viscoelastic coating [4]. The propagation of bending waves in a plate with viscoelastic

coatings in a simplified formulation is considered in [5, 6].

It is known that most of the information on the behavior of the waveguide is provided by the dispersion equation. Numerical analysis of the dispersion equations obtained during the investigation of the propagation of harmonic waves in a hereditarily elastic plate with two viscoelastic coatings is performed.

Taking into account the rheological properties of the material is accompanied by dispersion of the waves. The mechanisms by which the energy of elastic waves is converted into heat are not entirely clear. Different loss mechanisms are proposed [7, 8, 9], but not one of them does not fully meet all the requirements. Probably the most important mechanisms are internal friction in the form of sliding friction (or sticking, and then slipping) and viscous losses in pore fluids; the latter mechanism is most significant in strongly permeable rocks. Other effects that are probably generally less significant are the loss of some of the heat generated in the phase of compression of wave motion by thermal conductivity, piezoelectric and thermoelectric effects

and the energy going to the formation of new surfaces (which plays an important role only near the source).

Formulation of the problem.

We consider the propagation of harmonic waves in an infinite hereditarily elastic layered body, bounded by the planes $z = \pm h$ in the Cartesian coordinate system (Figure 1). The Oyz plane is compatible with the middle surface of the layer. We shall consider the propagation of waves in the direction of the x axis. Dynamic layer VAT will be described by the equations of motion for the case of a plane problem

$$\begin{aligned} \frac{\partial \sigma_{11}^{(n)}}{\partial x} + \frac{\partial \sigma_{13}^{(n)}}{\partial z} &= \rho_n \frac{\partial^2 v_1^{(n)}}{\partial t^2}; \\ \frac{\partial \sigma_{31}^{(n)}}{\partial x} + \frac{\partial \sigma_{33}^{(n)}}{\partial z} &= \rho_n \frac{\partial^2 v_3^{(n)}}{\partial t^2}, \quad (n=1,2,3) \end{aligned} \quad (1)$$

and the equations of state for hereditarily elastic material.

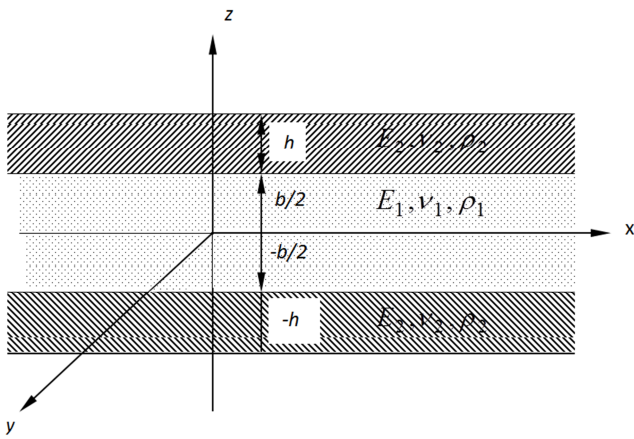


Figure 1. The design scheme.

In the present paper we take the equations of state in an integral operator form (the Rabotnov kernel [10])

$$\begin{aligned} \tilde{E}_n \frac{\partial v_1^{(n)}}{\partial x} &= \sigma_{11}^{(n)} - \tilde{\nu}_n (\sigma_{22}^{(n)} + \sigma_{33}^{(n)}); \\ \tilde{E}_n \frac{\partial v_3^{(n)}}{\partial z} &= \sigma_{33}^{(n)} - \tilde{\nu}_n (\sigma_{11}^{(n)} + \sigma_{22}^{(n)}); \\ 0 &= \sigma_{22}^{(n)} - \tilde{\nu}_n (\sigma_{11}^{(n)} + \sigma_{33}^{(n)}); \\ \frac{1}{2} \tilde{E}_n \left(\frac{\partial v_1^{(n)}}{\partial z} + \frac{\partial v_3^{(n)}}{\partial x} \right) &= (1 + \tilde{\nu}_n) \sigma_{13}^{(n)} \end{aligned} \quad (2)$$

In (1) and (2) the following notation is used: $\sigma_{ij}^{(n)}$ — components of the stress tensor, $v_i^{(n)}$ — displacement vector components,

$$\tilde{E}_n = E_n (1 - \Gamma_n^\bullet); \quad \tilde{\nu}_n = \nu_n + \frac{1 - 2\nu_n}{2} \Gamma_n^\bullet;$$

$$\Gamma_n^\bullet f(t) = m_n \int_{-\infty}^t \mathfrak{D}_{-1/2}^{(n)}(-\beta_n, t - \tau) f(\tau) d\tau. \quad (3)$$

Here E , ν — are the instantaneous Young's modulus and Poisson's ratio, m_n , β_n — parameters of the material. As the kernel of the integral operator we use the fractional exponential function of Rabotnov [12]

$$m_n \mathfrak{D}_{-1/2}^{(n)}(-\beta, t) = m_n t^{-1/2} \sum_{j=0}^{\infty} \frac{(-\beta_n)^j t^{j/2}}{\Gamma[(j+1)/2]},$$

where $\Gamma(j) = \int_0^\infty y^{j-1} \exp(-y) dy$ - gamma function.

In the case when the Koltunov-Rzhanitsen relaxation [19-22] core is used in place (3), then the elastic modulus is replaced by the operators, i.e. \tilde{E}_n — the elastic modulus of elasticity has the form [4,5]:

$$\tilde{E}_n \phi(t) = E_{0n} \left[\phi(t) - \int_0^t R_{En}(t - \tau) \phi(\tau) d\tau \right] \quad (4)$$

$\phi(t)$ — arbitrary time function; $R_{En}(t - \tau)$ — relaxation core; E_{01} — instantaneous modulus of elasticity; We assume the integral terms in (4) to be small, then the functions $\phi(t) = \psi(t) e^{-i\omega_R t}$, where $\psi(t)$ — a slowly varying function of time, ω_R — real constant. Then [13], we replace (4) by approximations of the form

$$\bar{E}_n \phi = E_{0j} \left[1 - \Gamma_j^C(\omega_R) - i \Gamma_j^S(\omega_R) \right] \phi, \quad (5)$$

where $\Gamma_n^C(\omega_R) = \int_0^\infty R_n(\tau) \cos \omega_R \tau d\tau$, $\Gamma_n^S(\omega_R) = \int_0^\infty R_n(\tau) \sin \omega_R \tau d\tau$,

respectively, the cosine and sine Fourier images of the relaxation core of the material. As an example of a viscoelastic material, we take three parametric relaxation nuclei $R_n(t) = A_n e^{-\beta_n t} / t^{1-\alpha_{jn}}$. On the influence function $R_n(t - \tau)$ The usual requirements of integrability, continuity (except for), signs - definiteness and monotonicity are imposed:

$$R > 0, \frac{dR(t)}{dt} \leq 0, 0 < \int_0^\infty R(t) dt < 1.$$

In the study of natural oscillations, we will investigate the properties of those modes (the modes are understood to mean particular solutions of the equations of motion in displacements that satisfy homogeneous boundary conditions on the face surfaces), which vary in time according to the harmonic law and satisfy the equations of motion (1), equations of state (2) and homogeneous boundary conditions on the faces:

- in the case of a rigid contact at the interface, the condition of continuity of the corresponding components of the stress tensor and

vector of displacement, i.e.

$$\sigma_{zz}^{(1)} = \sigma_{zz}^{(2)}; \quad \sigma_{xz}^{(1)} = \sigma_{xz}^{(2)}; \quad v_1^{(1)} = v_1^{(2)}; \quad v_3^{(1)} = v_3^{(2)}.$$

- If there is no friction at the interface,

$$\sigma_{zz}^{(1)} = \sigma_{zz}^{(2)}; \quad \sigma_{xz}^{(1)} = \sigma_{xz}^{(2)} = 0; \quad v_3^{(1)} = v_3^{(2)};$$

- On the free surface, the condition of freedom from stress is set, i.e.

$$\sigma_{zz}^{(1)} = 0; \quad \sigma_{zx}^{(1)} = 0, \quad (6)$$

where

$$\sigma_{xx}^{(n)} = \lambda_n \theta_n + 2\mu_n \frac{\partial v_1^{(n)}}{\partial x}; \quad \sigma_{xz}^{(j)} = \mu_j \left(\frac{\partial v_1^{(n)}}{\partial z} + \frac{\partial v_3^{(n)}}{\partial x} \right).$$

$$\sigma_{zz}^{(j)} = \lambda_j \theta_j + 2\mu_j \frac{\partial v_3^{(n)}}{\partial z} \quad \theta_j = \frac{\partial v_1^{(n)}}{\partial x} + \frac{\partial v_3^{(n)}}{\partial z}.$$

Thus, taking (5) into account, we arrive at the following system of equations:

$$\begin{aligned} \frac{\partial \sigma_{11}^{(n)}}{\partial x} + \frac{\partial \sigma_{13}^{(n)}}{\partial z} &= \rho_n \frac{\partial^2 v_1^{(n)}}{\partial t^2}; \\ \frac{\partial \sigma_{31}^{(n)}}{\partial x} + \frac{\partial \sigma_{33}^{(n)}}{\partial z} &= \rho_n \frac{\partial^2 v_3^{(n)}}{\partial t^2} \\ \bar{E}_n \frac{\partial v_1^{(n)}}{\partial x} &= \sigma_{11}^{(n)} - \nu_n (\sigma_{22}^{(n)} + \sigma_{33}^{(n)}); \\ \bar{E}_n \frac{\partial v_3^{(n)}}{\partial z} &= \sigma_{33}^{(n)} - \nu_n (\sigma_{11}^{(n)} + \sigma_{22}^{(n)}); \\ 0 &= \sigma_{22}^{(n)} - \nu_n (\sigma_{11}^{(n)} + \sigma_{33}^{(n)}); \\ \frac{1}{2} \bar{E}_n \left(\frac{\partial v_1^{(n)}}{\partial z} + \frac{\partial v_3^{(n)}}{\partial x} \right) &= (1 + \nu_n) \sigma_{13}^{(n)}, \end{aligned} \quad (7)$$

with boundary conditions (6).

2. Methods of Solution

The solution of the system of differential equations (2) - (6) (or (7) and (6)) for $\sigma_{ij}^{(n)}$ — components of the stress tensor and $v_i^{(n)}$ — the components of the displacement vector will be sought in the form

$$\begin{aligned} \sigma_{ij}^{(n)} &= \sigma_{ij}^n(z) \exp(i\omega t - ikx), \\ v_j^n &= V_j^n(z) \exp(i\omega t - ikx), \end{aligned} \quad (8)$$

where $\sigma_{ij}^n(z)$ и $V_j^n(z)$ — amplitude complex vector - function ($j=1,2$); k ($k = k_R + ik_I$) — wave number; C ($C = C_R + iC_I$) — complex phase velocity; ω — complex frequency.

To clarify their physical meaning, consider two cases:

1) $k = k_R$; $C = C_R + iC_I$, then the solution (8) has the form of a sinusoid with respect to x , the amplitude of which decays in time;

2) $k = k_R + ik_I$; $C = C_R$, Then at each point x the oscillations are steady, but with respect to x they decay.

In both cases, the imaginary parts k_I or C_I characterize the intensity of dissipative processes. Substituting relations (8) into a system of partial differential equations (2), we obtain a system of ordinary differential equations of the first order, solved with respect to the derivatives:

$$\begin{aligned} \frac{d\sigma_{13}^n}{dz} &= E_{4n}\sigma_{33}^n - E_{5n}V_1^n; \\ \frac{d\sigma_{33}^n}{dz} &= \delta_k \sigma_{31}^n - \Omega_n^2 V_3^n; \\ \frac{dV_3^n}{dz} &= E_{1n}\sigma_{33}^n - E_{2n}V_1^n; \\ \frac{dV_1^n}{dz} &= E_{3n}\sigma_{13}^n + \delta_k V_3^n, \end{aligned} \quad (9)$$

$$\begin{aligned} \sigma_{11}^n &= \frac{\nu_n^F}{1 - \nu_n^F} \sigma_{33}^n - \frac{E_n E_n^F (\delta + ik)}{1 - (\nu_n^F)^2} V_1^n; \\ \sigma_{22}^n &= \nu_n^F (\sigma_{11}^n + \sigma_{33}^n). \end{aligned}$$

where

$$\begin{aligned} E_{1n} &= \frac{1}{E_n E_n^F} (1 - (\nu_n^F)^2 - \frac{(\nu_n^F)^2 (1 + \nu_n^F)}{1 - \nu_n^F}); \quad E_{2n} = \frac{\delta_k}{1 - (\nu_n^F)^2}; \quad \delta_k = \delta + ik; \\ E_{3n} &= \frac{2(1 + \nu_n^F)}{E_n E_n^F}; \quad E_{4n} = \frac{\nu_n^F (\delta + ik)}{1 - \nu_n^F}; \quad E_{5n} = \frac{E_n E_n^F (\delta + ik)^2}{1 - (\nu_n^F)^2} + \rho_n \omega^2; \quad \Omega_n = \rho_n \omega^2. \\ E_n^F &= 1 - \frac{m_n}{\beta_n + \sqrt{i\omega}}; \quad \nu_n^F = \nu_n + \frac{1 - 2\nu_n}{2} \frac{m_n}{\beta_n + \sqrt{i\omega}}. \end{aligned}$$

At the interface ($z=h_n$) (in the case of a rigid contact), the condition of continuity of the corresponding components of the stress tensor and the displacement vector is set, i.e.

$$\begin{aligned} \sigma_{zz}^n(h_n) &= \sigma_{zz}^{n+1}(h_n); \quad \sigma_{xz}^n(h_n) = \sigma_{xz}^{n+1}(h_n); \\ V_1^n(h_n) &= V_1^{n+1}(h_n); \quad V_3^n(h_n) = V_3^{n+1}(h_n). \end{aligned}$$

- if at the interface ($z=h_n$) there is no friction, then

$$\sigma_{zz}^1 = \sigma_{zz}^2; \quad \sigma_{xz}^1 = \sigma_{xz}^2 = 0; \quad V_3^1 = V_3^2;$$

- on a free surface ($z=h_k$) the condition of freedom from stresses is set, i.e.

$$\sigma_{zz}^1 = 0; \quad \sigma_{zx}^1 = 0. \quad (10)$$

Thus, the spectral boundary value problem (9) - (10) is

formulated with respect to the parameter ω , which describes the propagation of waves in three-layer bodies. The posed spectral problem (7) - (8) in dimensionless variables is solved by the method of orthogonal sweep with a combination of the Muller method [23] on complex arithmetic:

$$\xi_n = \frac{x}{h_n}; \quad \zeta_n = \frac{z}{h_n}; \quad t^n = \frac{c_{2n}t}{h_n}, c_{2n} = \sqrt{\frac{E_n}{2(1+\nu_n)\rho_n}}.$$

2.1. Task 1

Consider a plate in which the neutral plane coincides with the plane x, z , and y coordinate in the direction of thickness, in the x direction, the plate is propagated by harmonic bending waves (Figure 1). The main material of the plate occupies the region $-\frac{b}{2} \leq z \leq \frac{b}{2}$, and for the covering, each of which have everywhere the same thickness h , occupies the region $\left(\frac{b}{2} \leq z \leq h, -h \leq z \leq -\frac{b}{2}\right)$. Let B_1 , μ_1 and ρ_1 respectively, the bulk modulus, the shear modulus, and the density of the plate material. Then the solution of the Rayleigh-Lembe problem for the base layer is obtained by methods of separating the variables and determining the displacements u, v, w (in the x, y and z directions, respectively) in the propagation of a harmonic wave with a frequency ω and a wave number k have the form [17]

$$\begin{aligned} u &= 0, \\ \vartheta &= (\alpha A_1 ch\alpha z - k C_1 sh\beta z) e^{i(\omega t - kx)}, \\ w &= (-ik A_1 ch\alpha z + i\beta C_1 sh\alpha z) e^{i(\omega t - kx)}, \end{aligned}$$

for $-\frac{b}{2} \leq y \leq \frac{b}{2}$, where A_1 and C_1 – constants, and the parameter α_1 and β_1 are defined by expressions

$$\alpha^2 = k^2 - \frac{\omega^2 \rho_1}{B_1 + \frac{4}{3}\mu_1}; \quad \beta^2 = k^2 - \frac{\omega^2 \rho_1}{\mu_1}. \quad (11)$$

On the basis of the ratios of the linear theory of elasticity between stresses and deformations of a homogeneous isotropic material, the plate has view

$$\begin{aligned} \sigma_{xz} &= (G - 2/3\mu) A_1 (\alpha^2 - k^2) sh\alpha z; \\ \sigma_{yy} &= A_1 \left[(G - 2/3\mu) (\alpha^2 - k^2) + 2\mu\alpha^2 \right] sh\alpha z - 2\mu k \beta C_1 sh\beta z; \\ \sigma_{xy} &= \sigma_{xx} = 0; \\ \sigma_{yz} &= -2\mu i k A_1 \alpha ch\alpha z + i\mu C_1 (k^2 + \beta^2) ch\beta z; \\ \sigma_{yy} &= A_1 \left[(G + 2/3\mu) (\alpha^2 - k^2) + 2\mu\alpha^2 \right] sh\alpha z + 2\mu k \beta C_1 sh\beta z. \end{aligned}$$

The expression for displacements of the points of the

upper outer covering, satisfying the equations of motion [1], generally have the form:

$$\begin{aligned} u &= 0; \\ \vartheta &= -\alpha c \left[\beta_1 sh\alpha_p (z-H) + F_1 ch\alpha_p (z-H) \right] + \\ &+ k \left[D_1 ch\beta_p (z-H) + M_1 sh\beta_p (z-H) \right]; \\ w &= ik \left[\beta_1 ch\alpha_p (z-H) + F_1 sh\alpha_p (z-H) \right] - \\ &- i\beta_c \left[D_1 sh\beta_p (z-H) + M_1 ch\beta_p (z-H) \right]; \end{aligned}$$

Here B_1, D_1, F_1, M_1 – constant, and the parameters α_p and β_p is determined by analogy with expressions (11) by the following dependences

$$\alpha_p^2 = k^2 - \frac{\omega^2 \rho_p}{G_p + \frac{4}{3}\mu_p}; \quad \beta_p^2 = k^2 - \frac{\omega^2 \rho_p}{\mu_p},$$

where the subscript c refers to the characteristics of the coating material. For the region $-H = -(h+b/2) \leq z \leq -b/2$ it is possible to measure an analogous family of dependencies using the antisymmetric properties of the flexural wave

$$u(z) = u(-z); \quad \vartheta(z) = \vartheta(-z); \quad w(z) = -w(-z) \quad (12)$$

Further, using properties (12) and restricting ourselves to the investigation of the domain $0 \leq z \leq H$. On the free surface $z = H$, the following boundary conditions must be satisfied:

$$\sigma_{xz} = \sigma_{zz} = \sigma_{yz} = 0.$$

In addition, on the surface of the contacts $z = b$, the continuity conditions of displacements must be satisfied

$$u(b^-) = u(b^+); \quad \vartheta(b^-) = \vartheta(b^+); \quad w(b^-) = w(b^+) \quad (13)$$

and the conditions of equality

$$\sigma_{xz}(b^-) = \sigma_{xz}(b^+); \quad \sigma_{zz}(b^-) = \sigma_{zz}(b^+); \quad \sigma_{yz}(b^-) = \sigma_{yz}(b^+).$$

On the basis of conditions (11), (12), (13) and the symmetry conditions, we form a system of six linear homogeneous algebraic equations with respect to unknown constant A_1, B_1, C_1, D_1, F_1 and M_1 ; the first two of these constants determine the deformed state of the plate, the other four – the deformed state of the upper layer of the coating. As a result, we obtain systems of homogeneous algebraic equations with complex coefficients $[C]\{q\} = \{0\}$. A system of homogeneous algebraic equations has a nontrivial solution, under the condition that the determinant of the basic matrix

$$[C] = 0 \quad (14)$$

where

$$\begin{aligned}
 C_{11} &= \alpha c h \alpha k; C_{12} = -k c h \beta k; C_{13} = \alpha_c s h \alpha_c (b-H); \\
 C_{14} &= \alpha_c c h \alpha_c (b-H); C_{15} = k c h \beta_c (b-H); \\
 C_{16} &= -k s h \beta (b-H); C_{21} = -k s h \alpha b; C_{22} = \beta s h \beta h \\
 C_{23} &= -k c h \alpha_c (b-H); C_{24} = -k c h \alpha_c (b-H); \\
 C_{25} &= \beta_c s h \beta_c (b-H); C_{26} = \beta s h \beta_c (b-H); \\
 C_{31} &= \alpha_1 s h \alpha b; C_{32} = -2 \mu k \beta s h \beta b \\
 C_{33} &= -\alpha_2 c h \alpha_c (b-H); C_{34} = -\alpha_2 c h \alpha_c (b-H); \\
 C_{35} &= 2 \mu_c k \beta_c s h \beta_c (b-H); C_{35} = 2 \mu_c k \beta_c c h \beta_c (b-H); \\
 C_{41} &= -2 \mu k \alpha c h \alpha h; C_{42} = \mu \alpha_3 c h \beta h; \\
 C_{43} &= -2 \mu_c k \alpha_c s h \alpha_c (b-H); C_{44} = -2 \mu_c k \alpha_c c h \alpha_c (b-H); \\
 C_{45} &= \mu_c \alpha_1 c h \beta_c (b-H); C_{46} = \mu_c \alpha_1 s h \beta_c (b-H); \\
 C_{51} &= C_{52} = C_{54} = C_{55} = 0; C_{53} = -\alpha_3; C_{56} = -2 \mu_c k \beta_c; \\
 C_{61} &= C_{62} = C_{63} = C_{66} = 0; C_{64} = -2 \mu_c k \alpha_c; C_{65} = \mu_c \alpha_c;
 \end{aligned}$$

where

$$\begin{aligned}
 \alpha_1 &= G(k^2 - \alpha^2) - \frac{2}{3} \mu(k^2 + \alpha^2); \alpha_2 = k^2 + \alpha^2; \\
 \alpha_3 &= G(k^2 - \alpha_c^2) - \frac{2}{3} \mu_c(k^2 + 2\alpha_c^2); \alpha_4 = k^2 + \beta_c^2.
 \end{aligned}$$

The transcendental equation (14) can be solved numerically by the method of Muller [7,23].

2.2. Task 2. The Propagation of Waves in a Viscoelastic Three-Layer Body on a Rigid Base

Suppose that in a Cartesian coordinate system a sequence of parallel planes is given. Suppose that the spaces between planes are filled with isotropic elastic media forming parallel layers. In the theoretical study of the described processes, we will assume that within each layer the wave propagation is described by the equations of elasticity theory (Lame). On the boundary of two bodies, the conditions of a rigid or sliding contact are set. With rigid contact at the interface, the condition of continuity of the corresponding components of the stress tensor and the displacement vector is established. On the free surface, the condition of freedom from stress is set. The problem is solved in the displacement potentials. The Sommerfeld radiation conditions satisfy the potential function at infinity. The dispersion equation has the form:

$$|\beta_{ij}| = 0, \quad i, j = 1, 2, \dots, 12 \quad (15)$$

where $\beta_{1,1} = (1 + \bar{s}_1^2) \exp(-k q_1 h)^*$; $\beta_{1,2} = (1 + \bar{s}_1^2) \exp(-k q_1 h_1^*)$

$$\begin{aligned}
 \beta_{1,3} &= -2 \exp(-k s_1 h)^*; \quad \beta_{1,4} = 2 \exp(k s_1 h^*); \\
 \beta_{1,5} &= \beta_{1,6} = 0; \quad \beta_{1,7} = \beta_{1,8} = 0; \quad \beta_{1,9} = \beta_{1,10} = \beta_{1,11} = \beta_{1,12} = 0; \\
 \beta_{2,1} &= -2 \bar{q}_1 \exp(-k q_1 h_1^*); \quad \beta_{2,2} = 2 \bar{q}_1 \exp(k q_1 h_1^*); \\
 \beta_{2,5} &= \beta_{2,6} = \beta_{2,7} = \beta_{2,8} = 0; \quad \beta_{2,9} = \beta_{2,10} = \beta_{2,11} = \beta_{2,12} = 0 \\
 \beta_{2,3} &= \left(\bar{s}_1 + \frac{1}{\bar{s}_1} \right) \times \exp(-k \bar{s}_1 h); \quad \beta_{2,4} = \left(\bar{s}_1 + \frac{1}{\bar{s}_1} \right) \times \exp(k \bar{s}_1 h); \\
 \beta_{3,1} &= \beta_{3,2} = \beta_{3,3} = \beta_{3,4} = 0; \quad \beta_{3,5} = \beta_{3,6} = \beta_{3,7} = \beta_{3,8} = 0; \\
 \beta_{3,9} &= (1 + \bar{s}_2^2) \exp(k q_2 h_2); \quad \beta_{3,10} = (1 + \bar{s}_2^2) \times \exp(-k q_2 h_2); \\
 \beta_{3,11} &= -2 \exp(k \bar{s}_{21} h_2); \quad \beta_{3,12} = 2 \exp(k \bar{s}_{21} h_2); \\
 \beta_{4,1} &= \beta_{4,2} = \beta_{4,3} = \beta_{4,4} = 0; \quad \beta_{4,5} = \beta_{4,6} = \beta_{4,7} = \beta_{4,8} = 0; \\
 \beta_{4,9} &= -2 \bar{q}_2 \exp(k \bar{q}_2 h_2); \quad \beta_{4,10} = 2 \bar{q}_2 \exp(-k \bar{q}_2 h_2); \\
 \beta_{4,11} &= (\bar{s}_2 + \bar{s}_2^{-1}) \exp(k \bar{s}_2 h_2); \quad \beta_{4,12} = (\bar{s}_2 + \bar{s}_2^{-1}) \exp(-k \bar{s}_2 h_2)
 \end{aligned}$$

The remaining elements (15) of the determinant are written in a similar way $h_1^* = h_1 + h_2$. Now this equation will be solved with respect to ω/k for different values of k . The roots of the equation are calculated on a computer with the following values of the dimensionless parameters:

$$C_{L1}^2 = 0.622; C_{L2}^2 = 3.360; C_{L3}^2 = 3.360; C_{S1}^2 = 0.776; C_{S2}^2 = 1.230;$$

$$C_{S3}^2 = 3.000; \mu_1 = 0.170; \mu_2 = 0.30; \mu_3 = 0.3.$$

3. Results

The results are plotted as dependency curves ($\xi = kH$, $H = h_1 + h_2$) from ω for $h_1^*/H = 0.5$. Let's consider two variants of the system. In the first variant, a homogeneous system is considered (Figure 4). coefficient behavior δ radically changed: dependence $\omega_1 \sim \xi$ became nonmonotonic. Of particular interest for practice is the minimum for fixing ξ value of the damping coefficient, the value of δ determines the damping properties as a whole.

In the case of a homogeneous system, δ is determined by the imaginary part of the complex natural frequency modulo the first. In the case of an inhomogeneous system ($R_{E2} = 0$) The role of the global damping coefficient is imaginary parts of both the first and second frequencies (Figure 5). The mathematical and physical aspects of this effect are explained in [13, 14, 15, 16]. The change in the parameter, on which the global coefficient of damping depends so substantially, can be achieved by varying the geometric dimensions or physical properties, thereby opening up the perspective possibility of effectively controlling the damping characteristics of dissipatively inhomogeneous viscoelastic systems.

The frequency equation (14) is solved by the Mueller method. In all cases, the Poisson's ratio of the middle layer is taken as 0.25; attitudes ρ_1/ρ densities of the coating materials and the middle layer are 0.35, and the ratio G_1/μ module of the volume compression of the coating material to the shear modulus of the material of the middle layer 0.20. The values given correspond to a system that consists of an elastic aluminum middle layer and a coating, is made of a typical

high-polymer material, and it is proposed that purely volumetric deformation of the coating during vibrations is not accompanied by energy dissipation. Various values of h/b were considered - the ratio of the thickness of the coating to the half-thickness of the middle layer and the different values of the coefficients γ - of the energy dissipation in the material under shear deformations. The results are presented in the form of graphs of the dependences of the real part $\Omega(\omega^2)$ and the coefficient of loss from the dimensionless wave number $h/x = kh/2\pi$ (the ratio of the half-thickness of the middle layer to the wavelength). The absolute error in calculating the eigenvalues (with respect to the exact equations) lies in the limit $14 \cdot 10^{-6}$. Figure 2 shows the change in the real and imaginary parts of the complex frequency as a function of the dimensionless wavelength. $\gamma = 0,35$ and the relative thickness of the coating $h/\lambda = 0,1$.

In the case of a dissipative homogeneous system δ (let's call it the global damping coefficient) is entirely determined by the imaginary part. First modulo complex natural frequency

$$\delta = \min(\Omega_{ik}), \quad k = 1, 2, \dots, n.$$

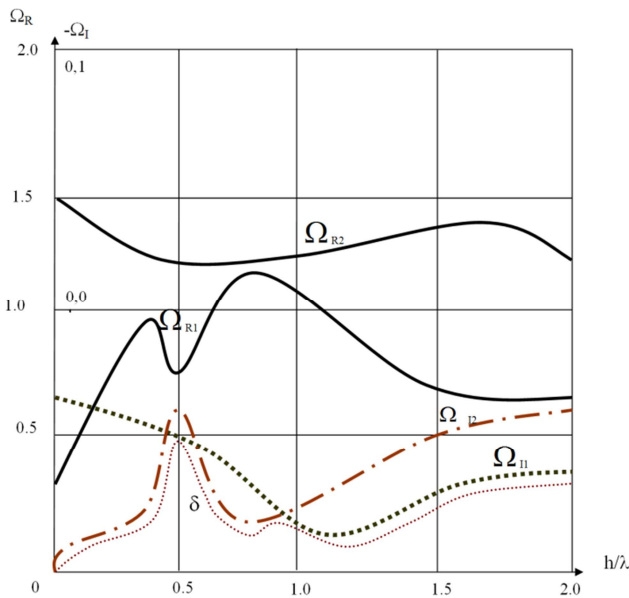
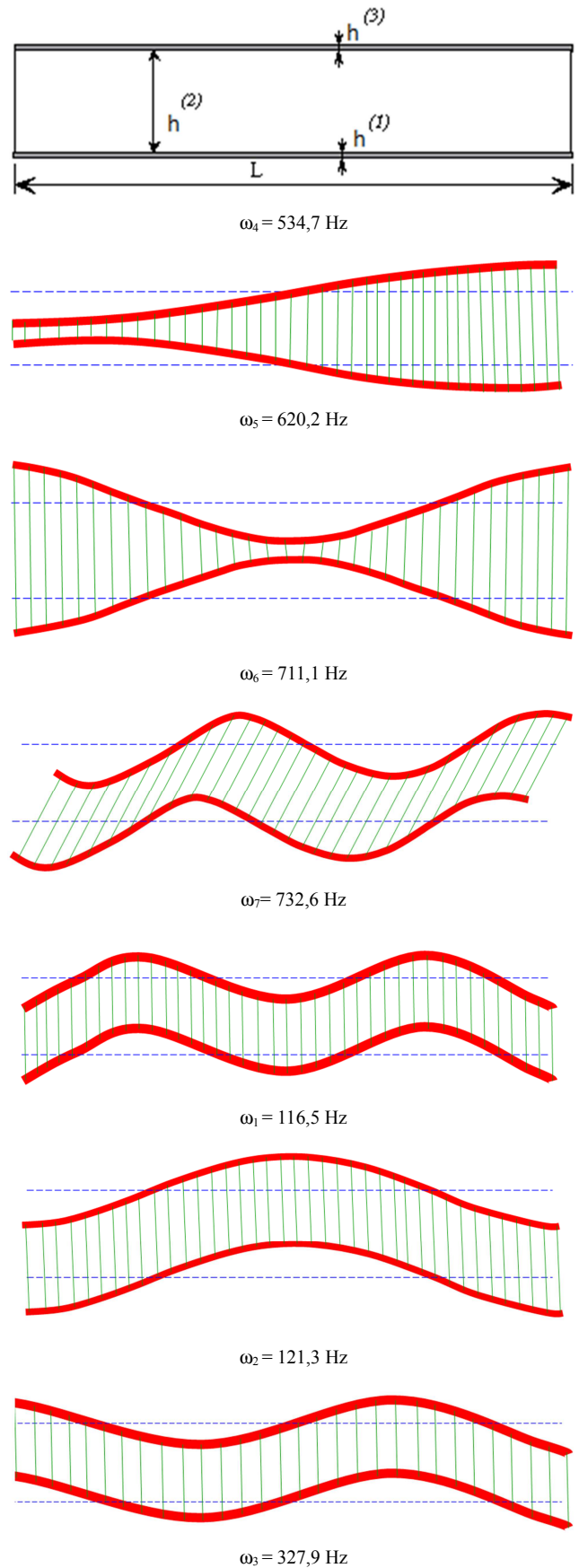


Figure 2. Dependence of natural frequencies (Ω_R) and the damping coefficients of the wavelength.

In the case of dissipatively inhomogeneous mechanical systems [5, 6], a nonmonotonic dependence of the damping coefficient on the wavelength is observed. The role of the global damping coefficient is played here by the imaginary parts of the first and second eigenfrequencies. When the corresponding frequencies approach (Ω_{Rk}) intersection of the imaginary parts of the first and second, the modes of the natural frequencies. A similar effect was observed when using the fractional-exponential kernel of the heredity Rabotnov Yu.N [24,25]. A similar effect was found in studies of systems with a finite number of degrees of freedom.



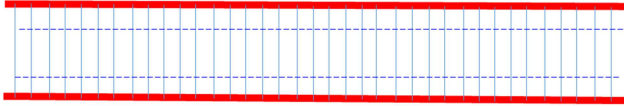


Figure 3. Forms of plate vibrations at different frequencies.

Figure 3 shows the change in the shape of the plate oscillations at different frequencies for $h=0,01m$, $E_2 = 5,5 \cdot 10^8 Pa$, $E_1 = 3,5 \cdot 10^{10} Pa$, $b = 0,08m$, $L=1m$, $\rho_1 = 27 kg/m^3$.

Rheological properties of the outer coating:

$$A = 0,048; \quad \beta = 0,05; \quad \alpha = 0,1.$$

4. Discussion

Analysis of dispersion equations and their numerical solutions allows us to draw the following conclusions:

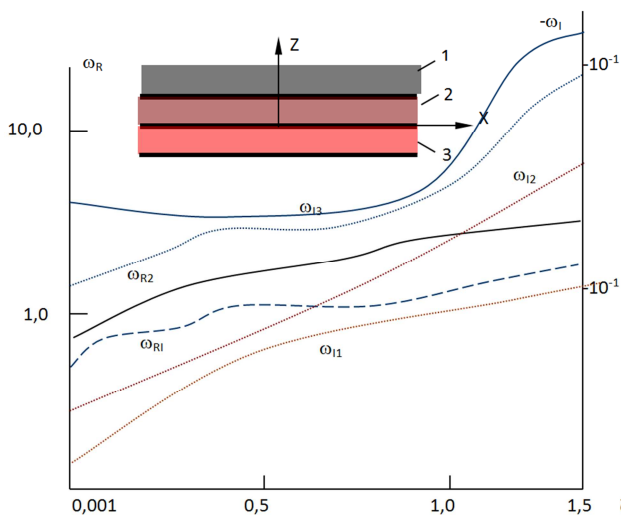


Figure 4. Change of natural frequencies from the wave number (dissipative homogeneous system).

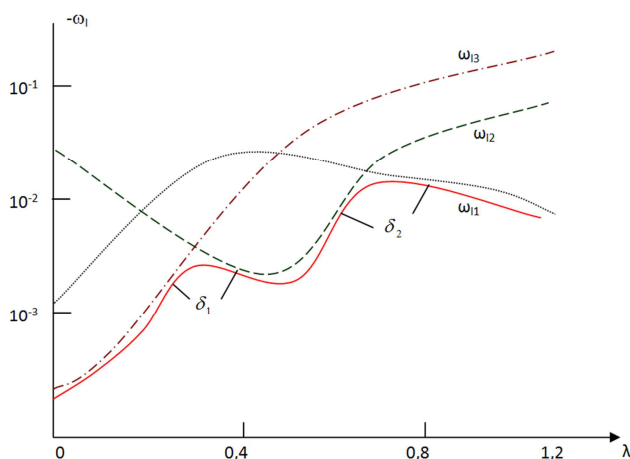


Figure 5. Change of natural frequencies from the wave number (dissipatively inhomogeneous system).

- for the dissipatively inhomogeneous mechanical systems, the "Trojanovskii-Safarov" effect [18] was found: the

nonmonotonic dependence of the damping coefficients on the geometric and physico-mechanical parameters of mechanical systems;

- there is a symmetry of the dispersion curves when the complex wave number \tilde{k} on $-\tilde{k}$;

- the larger the value of the parameter of the fractional exponential parameter of the nucleus m and (or less the value β), the earlier the dispersion curves with the positive and negative imaginary parts begin to diverge \tilde{k} ;

- with a decrease in the values of m and (or) with increasing values β the behavior of the dispersion curves tends to the elastic case;

- dispersion curves of the hereditary-elastic spectrum corresponding to the real branches of the elastic spectrum are complex with a positive imaginary part \tilde{k} , which determines the attenuation of the coordinate solution;

- in the vicinity of the locking frequencies of the elastic spectrum, the branches of the hereditary-elastic spectrum have the greatest curvature. Increasing the values of m , like decreasing the values β , leads to a smoothing of the dispersion curves in these regions. Thus, the elastic spectrum can approximately be regarded as asymptotic for the hereditarily elastic $k \rightarrow 0$, $\beta > 1$.

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